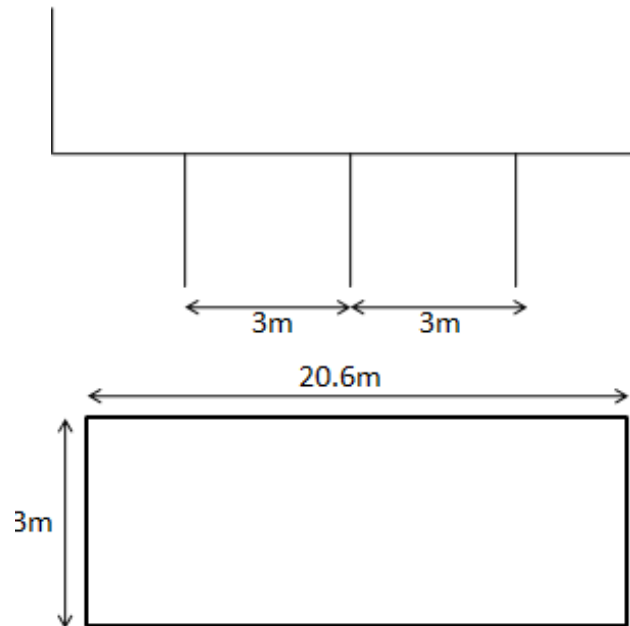


Design of X- Beam



Depth of beam	=	1.4 m
Thickness of beam	=	0.4 m
Density of concrete, γ_c	=	25 kN/m
Thickness of slab	=	0.25 m
Thickness of W.C.	=	0.1 m
Density of W.C.	=	22 kN/m
Spacing of X – beam	=	20.6 m

Step1:-

Dead Load:

Spacing of Main girder (Span of X - beam, l_x) =

3 m

Weight of rib of x- beam = $1.4 \times 0.4 \times 25$

= 14 kN/m

Weight of (Slab + W.C.) = $0.25 \times 25 + 0.1 \times 22$

= 8.45 kN/mm²

Total load of deck slab = $8.45 \times 2 \times 0.5 \times 3 \times 3/2$

= 38.025 kN

This load is assumed uniformly distributed along ?

3-meter length

Load per meter run due to deck slab

= $38.025/3$

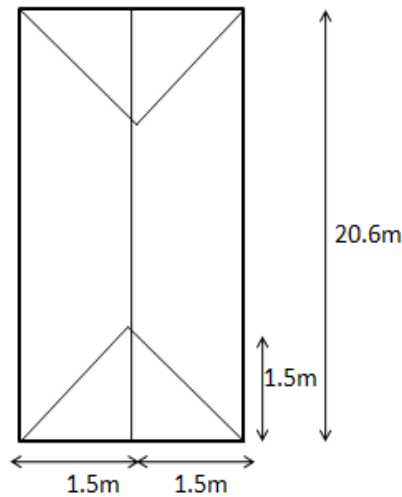
= 12.7 kN/m

Total dead load per meter run = $14 + 12.7$

= 26.7 kN/m

This load is assumed uniformly distributed along

This cross – girder is assumed to be rigid.



The reaction on each longitudinal girder

$$= 26.7 \times 2 \times 3 / 3$$

$$= 53.4 \text{ kN} \quad ?$$

Step2:- Live Load:

B.M. due to dead load at a distance from support ? = 1.975 m
 Class AA (tracked) vehicle produce maximum B.M. and S.F. in x- girder.

This position of loading longitudinal direction gives maximum reaction to x – girder.

$$\text{Magnitude of reaction} = 700 \times 19.7 / 20.6$$

$$670 \text{ kN}$$

$$\text{No. of long. Beam} = 3$$

Because x-girder is rigid, reaction on each longitudinal girder

$$= 700 / 3$$

$$233.33 \text{ kN}$$

Maximum bending moment occurs under track load,

$$= 700 \times 1.975 / 3$$

$$460.83 \text{ kN/m}$$

$$\text{Impact factor} = 1.1$$

Including, Impact factor for B.M. due to live load

$$= 460.83 \times 1.1$$

$$= 507 \text{ kN/m}$$

B.M. due to dead load at a distance of 1.975 from support ?

$$= 53.4 \times 1.975 - (26.675 \times (1.975/2)^2)$$

$$= 80 \text{ kN/m}$$

$$\text{Total Bending Moment} = 507 + 80$$

$$= 587 \text{ kN/m}$$

Step3:- Live Load shear including Impact = 700 * 1.1 / 3

$$= 257 \text{ kN}$$

$$\text{Dead Load shear} = 53.35 \text{ kN}$$

$$\text{Total shear for depth} = 53.35 + 257$$

$$= 310.35 \text{ kN}$$

	Clear cover	=	40 m	? Take on tmp
	Diameter	=	25 m	?
Step4:-	Effective depth	=	1650-(40+25/2)	
		=	1597.5 mm	
		=	0.1598 m	
Step5:-	Check for shear:			
	b	=	400 mm	
	τ_v	=	310.35*1000/400*1597.5	
		=	0.486 N/mm ²	
	Spacing of 2 – legged stirrups			
	Diameter of bar used	=	12 mm	
	Asv	=	2*0.785*12 ²	
		=	226 mm ²	
	σ_{sv}	=	200	
	Spacing of stirrups Sv	=	200*226*1597.5/310.35*1000	
			232.7 mm ²	
	Area of steel required :-			
	Ast	=	587*10 ⁶ /0.9*1597.5*200	
		=	2041.4 mm ²	
	Minimum shear reinforcement	=	226/400*.0015	
		=	377 mm ²	

sketch?

/ other sheet

Design of Bearing Pad

Given

Max. Dead Load	=	1025kN
f_{ck}	=	30
Thickness	=	75mm
Live load	=	365+75
	=	440kN
Horizontal force due to live load	=	80 kN
Assumed Size of bearing pad		
Breadth of pad(bp)	=	550mm
Length of pad(Lp)	=	950 mm
Side cover(Sc)	=	6 mm
Thickness of steel	=	10 mm

Step 1-

Thickness should be between Breadth of pad(bp)/10 to Length of pad(Lp)/5
 55 to 110
 O.K

Step 2-

Live load	=	400kN
Loaded area	=	$(bp * Lp) - (2(bp + Lp) * Sc)$
		504500mm ²
Total load (Nmax)	=	DL+LL
	=	1465kN
Approx.	~	1500kN
Nmin	=	1025 kN
A1	=	4
A2	=	2
A1/A2	=	2

Step 3- Grade Provided M30 :-

Allowable contact pressure	=	$0.25 \times f_{ck} \times \sqrt{\frac{A1}{A2}}$
		10.61 Mpa
Effective area of bearing required	=	$1500 * 1000 / 10.61$
	=	14.1376mm ²
σ_m	=	total load/loaded area
	=	2.973 Mpa

Step 4- Thickness of individual Elastomer layer

hi	=	15 mm
No.	=	5
Thickness of steel Laminates	=	10 mm
Overall thickness of bearing	=	75 mm
Side cover	=	6 mm
Total thickness of elastomer(t)	=	55 mm
Shear modulus assumed, d	=	1 N/mm ²
Shear strain due to creep, shrinkage, temperature(L)	=	0.0005
From temp. sheet(K)	=	41000
Shear strain per bearing due to creep, shrinkage, temperature	=	$(L * K) / 2t$
	=	0.186
Shear strain due to longitudinal force	=	$80 * 1000 / 504500$
	=	0.159
Shear strain due to translation	=	B/loaded area

		=	0.345	Safe
Step 5-	Calculation of rotation,			
	σ_{min}	=	$0.5 \cdot \sigma_m \cdot h_i / b \cdot s^2$	
	N	=	538	
	O	=	938	
	s	=	15	
(I)	Shape factor (s)	=	Loaded area / (2(N+O)h _i)	
		=	11.393	safe
(ii)	Assume, $\sigma_m, max.$	=	10 MPa	
	$\alpha_{b_i}, max.$	=	$0.5 \cdot \sigma_m \cdot h_i / b \cdot s^2$	
		=	0.00107 radians	
	P	=	2.973	
	β	=	P/10	
		=	0.2973 MPa	
	Permissible rotation	=	$\beta \cdot N \cdot \alpha_{b_i}, max.$	
		=	0.00159 MPa	
Step 6-	Friction			
	Shear strain(Z)	=	0.345 MPa	
Check:-		=	$0.2 + 0.1 \cdot \sigma_m$	
		=	0.4973	safe
	where,	σ_m	=	2.973
Check:-			$2 \text{MPa} < \sigma_m < 10 \text{MPa}$	satisfied
	Total Shear Stress			
Step 7-	Shear stress due to compression(X)	=	$(1.5 \cdot \sigma_m) / s$	
		=	0.3914 MPa	
	Shear Stress due to Horizontal deformation(Y)	=	$0.5 \cdot b / h_i^2 \cdot \alpha_{b_i}$	
		=	0.688 MPa	
	Shear Stress due to Horizontal rotation	=	X+Y+Z	
		=	1.4244 MPa	safe

	Bearing pad	beam
A	365	700
	75	19.7
B	80*1000	20.6
		1.975
		26.675
		40
		25
		1650
		200
		0.9
		0.0015